Applied Mathematics 205

Unit V: Eigenvalue Problems

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Unit V: Eigenvalue Problems Chapter V.1: Motivation

A matrix $A \in \mathbb{C}^{n \times n}$ has eigenpairs $(\lambda_1, v_1), \ldots, (\lambda_n, v_n) \in \mathbb{C} \times \mathbb{C}^n$ such that

$$Av_i = \lambda v_i, \quad i = 1, 2, \dots, n$$

(We will order the eigenvalues from smallest to largest, so that $|\lambda_1| \le |\lambda_2| \le \cdots \le |\lambda_n|$)

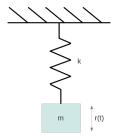
It is remarkable how important eigenvalues and eigenvectors are in science and engineering!

For example, eigenvalue problems are closely related to resonance

- Pendulums
- Natural vibration modes of structures
- Musical instruments
- Quantum mechanics
- Lasers
- Nuclear Magnetic Resonance (NMR)

► ...

Consider a spring connected to a mass



Suppose that:

- the spring satisfies Hooke's Law, F(t) = ky(t)
- a periodic forcing, r(t), is applied to the mass

¹Here y(t) denotes the position of the mass at time t

Then Newton's Second Law gives the ODE

$$y''(t) + \left(\frac{k}{m}\right)y(t) = r(t)$$

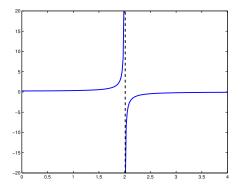
where $r(t) = F_0 \cos(\omega t)$

Recall that the solution of this non-homogeneous ODE is the sum of a homogeneous solution, $y_h(t)$, and a particular solution, $y_p(t)$

Let
$$\omega_0 \equiv \sqrt{k/m}$$
, then for $\omega \neq \omega_0$ we get:²
 $y(t) = y_h(t) + y_p(t) = C \cos(\omega_0 t - \delta) + \frac{F_0}{m(\omega_0^2 - \omega^2)} \cos(\omega t)$,

 $^{^2 {\}it C}$ and δ are determined by the ODE initial condition

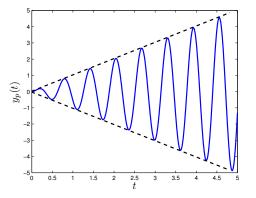
The amplitude of $y_p(t)$, $\frac{F_0}{m(\omega_0^2-\omega^2)}$, as a function of ω is shown below



Clearly we observe singular behavior when the system is forced at its natural frequency, i.e. when $\omega = \omega_0$

Motivation: Forced Oscillations

If we solve the ODE in the case that $\omega = \omega_0$, we obtain $y_p(t) = \frac{F_0}{2m\omega_0} t \sin(\omega_0 t)$



The solution is unbounded as $t \to \infty$, this is resonance

In general, the natural frequency ω_0 is the frequency at which the unforced system has a non-zero oscillatory solution

To calculate ω_0 directly, we substitute an oscillatory "ansatz" into the unforced equation and solve for the frequency

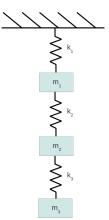
For example, for the single spring-mass system we substitute³ $y(t) \equiv xe^{i\omega_0 t}$ into $y''(t) + (\frac{k}{m})y(t) = 0$

This gives a scalar equation for ω_0 :

$$kx = \omega_0^2 mx \implies \omega_0 = \sqrt{k/m}$$

³Here x is the amplitude of the oscillatory solution

Suppose now we have a spring-mass system with three masses and three springs



In the unforced case, this system is governed by the ODE system

$$My''(t) + Ky(t) = 0,$$

where M is the mass matrix and K is the stiffness matrix

$$M \equiv \left[\begin{array}{ccc} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{array} \right], \quad K \equiv \left[\begin{array}{ccc} k_1 + k_2 & -k_2 & 0 \\ -k_2 & k_2 + k_3 & -k_3 \\ 0 & -k_3 & k_3 \end{array} \right]$$

We again seek a nonzero oscillatory solution to this ODE, i.e. set $y(t) = xe^{i\omega t}$, where now $y(t) \in \mathbb{R}^3$

This gives the algebraic equation

$$Kx = \omega^2 Mx$$

Setting $A \equiv M^{-1}K$ and $\lambda = \omega^2$, this gives the eigenvalue problem

$$Ax = \lambda x$$

Here $A \in \mathbb{R}^{3 \times 3}$, hence we obtain natural frequencies from the three eigenvalues λ_1 , λ_2 , λ_3

The spring-mass systems we have examined so far contain discrete components

But the same ideas also apply to continuum models

For example, the wave equation models vibration of a string (1D) or a drum (2D)

$$\frac{\partial^2 u(x,t)}{\partial t^2} - c\Delta u(x,t) = 0$$

As before, we write $u(x,t) = \tilde{u}(x)e^{i\omega t}$, to obtain

$$-\Delta \tilde{u}(x) = \frac{\omega^2}{c}\tilde{u}(x)$$

which is a PDE eigenvalue problem

We can discretize the Laplacian operator with finite differences to obtain an algebraic eigenvalue problem

$$A\mathbf{v} = \lambda \mathbf{v},$$

where

- ▶ the eigenvalue $\lambda = \omega^2/c$ gives a natural vibration frequency of the system
- the eigenvector (or eigenmode) v gives the corresponding vibration mode

We will use the Matlab functions eig and eigs to solve eigenvalue problems:

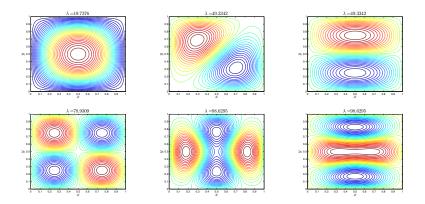
- eig: find all eigenvalues/eigenvectors of a dense matrix
- eigs: find a few eigenvalues/eigenvectors of a sparse matrix

Matlab demo: Eigenvalues/eigenmodes of Laplacian on $[0, 1]^2$, zero Dirichlet boundary conditions

Based on separation of variables, we know that eigenmodes are $sin(\pi ix) sin(\pi jy)$, i, j = 1, 2, ...

Hence eigenvalues are $(i^2 + j^2)\pi^2$

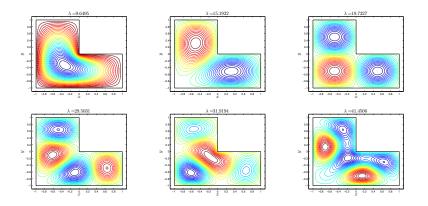
i	j	$\lambda_{i,j}$
1	1	$2\pi^2pprox 19.74$
1	2	$5\pi^2pprox$ 49.35
2	1	$5\pi^2pprox$ 49.35
2	2	$8\pi^2pprox 78.96$
1	3	$10\pi^2pprox98.97$
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In general, for repeated eigenvalues, computed eigenmodes are linearly independent members of the corresponding eigenspace

e.g. eigenmodes corresponding to $\lambda = 49.3$ are given by $\alpha \sin(\pi x) \sin(\pi 2y) + \beta \sin(\pi 2x) \sin(\pi y), \quad \alpha, \beta \in \mathbb{R}$

And of course we can compute eigenmodes of other shapes...



An interesting mathematical question related to these issues: "Can one hear the shape of a drum?"⁴

The eigenvalues for a shape in 2D correspond to the resonant frequences that a drumhead of that shape would have

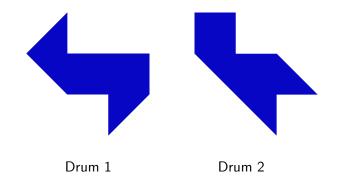
Therefore, the eigenvalues determine the harmonics, and hence the sound of the drum

So in mathematical terms, this question is equivalent to: If we know all of the eigenvalues, can we uniquely determine the shape?

⁴Posed by Mark Kac in American Mathematical Monthly in 1966

It turns out that the answer is no!

In 1992, Gordon, Webb, and Wolpert constructed two different 2D shapes that have exactly the same eigenvalues!



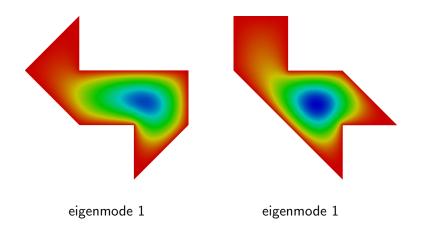
We can compute the eigenvalues and eigenmodes of the Laplacian on these two shapes using the algorithms from this ${\rm Unit}^5$

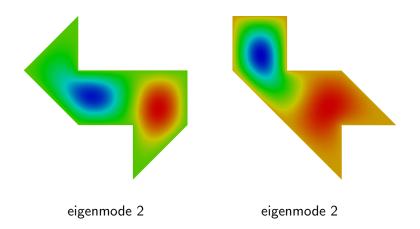
The first five eigenvalues are computed as:

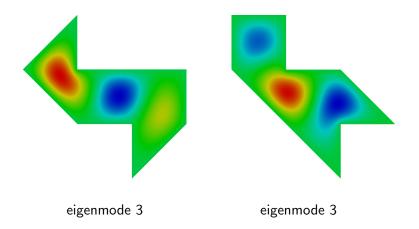
	Drum 1	Drum 2
λ_1	2.54	2.54
λ_2	3.66	3.66
λ_3	5.18	5.18
λ_4	6.54	6.54
λ_5	7.26	7.26

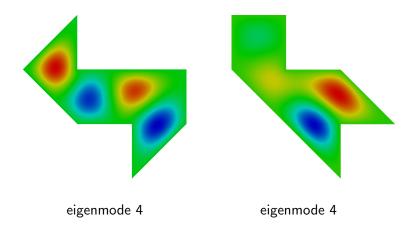
We next show the corresponding eigenmodes...

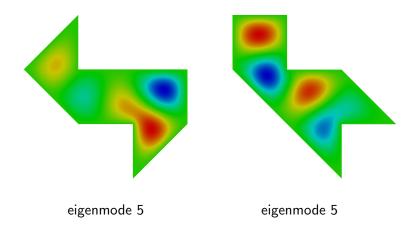
⁵Note here we employ the Finite Element Method (outside the scope of AM205), an alternative to F.D. that is well-suited to complicated domains











Summary

Eigenvalue problems have many interesting and important applications in science and engineering

In practice, the main challenge is often to formulate a problem as $Ax = \lambda x$

We can then employ reliable and efficient algorithms for computing the eigenvalues/eigenvectors (e.g. eig, eigs in Matlab)

In the remaining chapters of this Unit we explore the mathematical ideas that underpin algorithms for eigenvalue problems